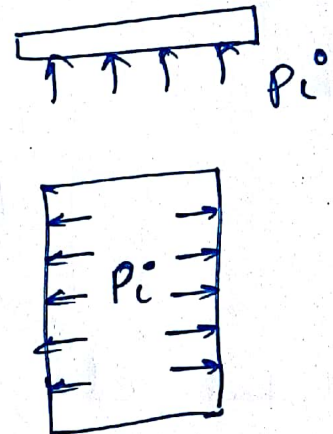
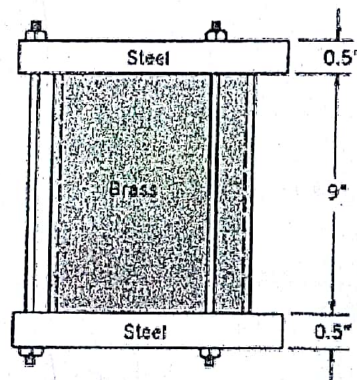
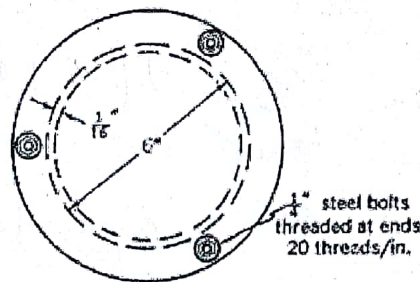


Name: \_\_\_\_\_ Roll no. \_\_\_\_\_  
 Department of Aerospace Engineering, Indian Institute of Technology, Madras.  
 AS 2010: Basic strength of materials. Quiz 8

1. A small experimental pressure vessel is made from a 9 inch long brass cylinder ( $E = 15 \times 10^6$  psi [pounds per square inch]) of 6 inch mean diameter, and 1/16 inch wall thickness, and two 1/2 inch thick steel plates held together by three 1/4 inch diameter steel bolts set on a 7 inch diameter bolt circle. The vessel is put together with nuts on the three bolts brought up snug. Then, each nut is tightened one-half turn additional. Estimate the internal pressure in psi at which the vessel is certain to leak.



No. of bolts - 3

$$\text{Total axial load} = P_i \times \frac{\pi}{4} (d_m)^2$$

↑ mean dia.

$$= \frac{\pi}{4} \times 36 \times P_i$$

$$\text{force acting in each bolt } (F_b) = \frac{\text{Total axial load}}{3} = \frac{3\pi P_i}{3}$$

$$\text{Stress on bolt} = \sigma_b = \frac{F_b}{A_{\text{bolt}}} = \frac{F_b}{A_{\text{steel}}}$$

$$\sigma_b = \frac{3\pi P_i}{\pi/4}$$

$$\sigma_b = \frac{3\pi P_i}{\pi/4}$$

Total elongation of bolt  $\epsilon_b = E_b \cdot L_{\text{bolt}}$

Length of bolt =  $9 + 0.5 + 0.5 = 10''$

$$E_b = \frac{\delta_b}{E_{\text{steel}}} = \frac{3\pi P_i^0}{\frac{\pi}{4} \left(\frac{1}{4}\right)^2 \times E_{\text{steel}}} = \frac{192 P_i^0}{E_{\text{steel}}}$$

$$\delta_b = \frac{192 P_i^0 \times 10''}{E_{\text{steel}}}$$

Because of poisson effect, brass cylinder will contract.

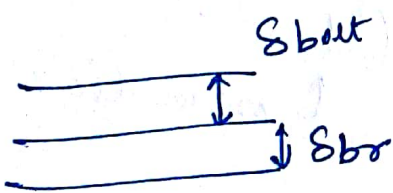
$$\epsilon_{\text{br}} = \frac{\delta_z}{E_{\text{br}}} - \nu \left( \frac{\delta_\theta + \delta_r}{E_{\text{br}}} \right)$$

Circumferential stress in cylinder  $(\delta_\theta) = \left( \frac{P_i d}{2t} \right)$ .

$$\delta_z = 0; \delta_r = 0$$

$$E_{\text{br}} = - \frac{\nu_{\text{br}} P_i d}{2 E_{\text{br}} t}$$

Total elongation =  $E_{\text{br}} \cdot L_{\text{br}} = \frac{-\nu_{\text{br}} P_i d}{2 E_{\text{br}} t} \times 9''$



Compatibility equation

$$\delta_{\text{br}} + \delta_{\text{bolt}} = \frac{1}{2} \left( \frac{1}{20} \right)''$$

$E_{\text{steel}} = 200 \text{ GPa}$   
 Convert into  
 Psi  
 $= 29 \times 10^6 \text{ Psi}$

$$\frac{192 P_i^0 \times 10''}{29 \times 10^6} + \frac{0.31 \times P_i \times 6 \times 9''}{2 \times 15 \times 10^6 \times \left(\frac{1}{16}\right)} = \frac{1}{40}''$$

$$P_i^0 = 332.7 \text{ Psi}$$

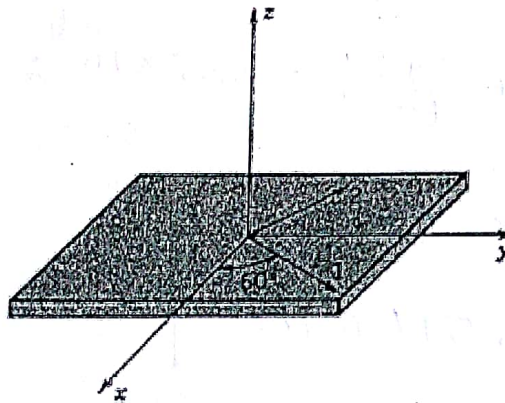
Name: \_\_\_\_\_ Roll no. \_\_\_\_\_  
 Department of Aerospace Engineering, Indian Institute of Technology, Madras.  
 AS 2010: Basic strength of materials. Quiz 8

2. The principal strains in the plane of a thin flat aluminium plate ( $E = 70 \text{ GPa}$ ,  $\nu = 0.3$ ), which is loaded in its plane are

$$\epsilon_1 = 320 \times 10^{-6}, \text{ and}$$

$$\epsilon_2 = -540 \times 10^{-6}.$$

Find the stresses  $\sigma_{xx}$ ,  $\sigma_{yy}$  and  $\sigma_{xy}$ , where the  $x$  and  $y$  axes are located as shown.



Principal strains are given.

$$\epsilon_{1,2} = \frac{\epsilon_x + \epsilon_y}{2} \pm \sqrt{\left(\frac{\epsilon_x - \epsilon_y}{2}\right)^2 + \left(\frac{\gamma_{xy}}{2}\right)^2} \quad \leftarrow$$

$$\theta = \frac{1}{2} \tan^{-1} \left( \frac{\gamma_{xy}}{\epsilon_x - \epsilon_y} \right) \quad \left| \quad \begin{array}{l} 3 \text{ unknown} \\ 3 \text{ equations} \end{array} \right.$$

$$2 \times 60^\circ = \tan^{-1} \left( \frac{\gamma_{xy}}{\epsilon_x - \epsilon_y} \right); \quad \gamma_{xy} = \sqrt{3} (\epsilon_y - \epsilon_x) \quad \text{--- (1)}$$

$\epsilon_x \neq$  Substitute  $\gamma_{xy}$  value in principal strains equation.

$$\epsilon_1 = \frac{3\epsilon_x - \epsilon_y}{2} \quad \& \quad \epsilon_2 = \frac{3\epsilon_y - \epsilon_x}{2}$$

$$320 \times 10^{-6} = \frac{3\varepsilon_x - \varepsilon_y}{2}$$

$$-540 \times 10^{-6} = \frac{3\varepsilon_y - \varepsilon_x}{2}$$

$$\varepsilon_x = 105 \times 10^{-6}$$

$$\varepsilon_y = -325 \times 10^{-6}$$

$$\tau_{xy} = -430\sqrt{3} \times 10^{-6}$$

$$\varepsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} ; 105 \times 10^{-6} = \frac{\sigma_x}{70 \times 10^9} - \frac{0.3}{70 \times 10^9} \sigma_y \quad \text{--- (a)}$$

$$\varepsilon_y = -\nu \frac{\sigma_x}{E} + \frac{\sigma_y}{E} ; -325 \times 10^{-6} = -\frac{0.3}{70 \times 10^9} \sigma_x + \frac{\sigma_y}{70 \times 10^9} \quad \text{--- (b)}$$

from a & b

$$\sigma_x = 0.577 \text{ MPa}$$

$$\sigma_y = -22.577 \text{ MPa}$$

$$\tau_{xy} = \tau_{xy} \times G$$

$$= -430 \times \sqrt{3} \times 10^{-6} \times \frac{70 \times 10^9}{2(1+0.3)}$$

$$\tau_{xy} = -20.05 \text{ MPa}$$